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The following Listing of Claims will replace all prior versions, and listings, of claims in the application.

LISTING OF CLAIMS:

1. (Currently Amended) An air conditioning system (1) (101) (201) (401) (601) configured to treat the <u>a</u> latent heat load and the <u>a</u> sensible heat load in a room by performing a vapor compression refrigeration cycle operation, comprising:

a plurality of first utilization side refrigerant circuits (10a, 10b) (110a, 10b) (210a, 210b) (310a, 2310b) (410a, 2410b) (510a, 2510b) (610a, 2610b) (710a, 210b) (910a, 910b) each having an adsorbent heat exchanger (22, 23, 32, 33) (122, 123, 132, 133) (322, 323, 332, 333) (522, 523, 532, 533) (722, 723, 732, 733) (922, 923, 932, 933) provided with an adsorbent on the a surface each thereof, eapable of configured for alternating between an adsorption process in which moisture in air is adsorbed onto the adsorbent by causing the adsorbent heat exchanger to function as an evaporator that evaporates refrigerant and a regeneration process in which moisture is desorbed from the adsorbent by causing the adsorbent heat exchanger to function as a condenser that condenses the refrigerant, and connected in parallel with one another; and

a plurality of second utilization side refrigerant circuits (10c, 10d) (110c, 110d) (210c, 210d) (310c, 310d) (410c, 410d) (510c, 510d) (610c, 610d) (710c, 710d) (1010a, 1010b) each having an air heat exchanger, configured for (42, 52) (142, 152) (242, 252) (342, 352) (442, 452) (542, 552) (642, 652) (742, 752) (1022, 1032), capable of exchanging heat between refrigerant and air, and connected in parallel with one another, wherein

the air conditioning system can first utilization side refrigerant circuits being configured to supply a room with air that passed through the adsorbent heat exchanger, and

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ean the second utilization side refrigerant circuits being configured to supply a room with air that passed through the air heat exchangers.

2. (Currently Amended) The air conditioning system (1) (101) (201) (301) (401) (501) (601) (701) according to claim 1, <u>further</u> comprising[[:]]

a heat source side refrigerant circuit (10e) (110e) (210e) (310e) (410e) (510e) (610e) (710e) including a compression mechanism (61) (161) (261) (361) (461) (561) (661) (761) and a heat source side heat exchanger, the heat source side refrigerant circuit being (63) (163) (263) (363) (463) (563) (663) (763) and used as a heat source by both the first utilization side refrigerant circuits (10a, 10b) (110a, 110b) (210a, 210b) (310a, 310b) (410a, 410b) (510a, 510b) (610a, 610b) (710a, 710b) and the said second utilization side refrigerant circuits (10e, 10d) (110e, 110d) (210e, 210d) (310e, 310d) (410e, 410d) (510e, 510d) (610e, 610d) (710e, 710d), wherein

the first utilization side refrigerant circuits are being connected to a discharge gas connection pipe (8) (108) (208) (308) (408) (508) (608) (708) connected to a discharge side of the compression mechanism, and are being connected to an inlet gas connection pipe (9) (109) (209) (309) (409) (509) (609) (709) connected to an inlet side of the compression mechanism.

3. (Currently Amended) An air conditioning system (1) (101) (201) (301) (401) (501) (601) (701) configured to treat the <u>a</u> latent heat load and the <u>a</u> sensible heat load in a room by performing a vapor compression refrigeration cycle operation, the air conditioning system comprising:

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a first utilization side refrigerant circuit (10a, 10b) (110a, 110b) (210a, 210b) (310a, 310b) (410a, 410b) (510a, 510b) (610a, 610b) (710a, 710b) having an adsorbent heat exchanger (22, 23, 32, 33) (122, 123, 132, 133) (322, 323, 332, 333) (522, 523, 532, 533) (722, 723, 732, 733) provided with an adsorbent on the a surface each thereof and eapable of configured for alternating between an adsorption process in which moisture in air is adsorbed onto the adsorbent by causing the adsorbent heat exchanger to function as an evaporator that evaporates refrigerant, and a regeneration process in which moisture is desorbed from the adsorbent by causing the adsorbent heat exchanger to function as a condenser that condenses refrigerant;

a plurality of second utilization side refrigerant circuits (10e, 10d) (110e, 110d) (210e, 210d) (310e, 310d) (410e, 410d) (510e, 510d) (610e, 610d) (710e, 710d) each having an air heat exchanger (42, 52) (142, 152) (242, 252) (342, 352) (442, 452) (542, 552) (642, 652) (742, 752), eapable of configured for exchanging heat between refrigerant and air, and connected in parallel with one another; and

a heat source side refrigerant circuit (10e) (110e) (210e) (310e) (410e) (510e) (610e) (710e) having including a compression mechanism (61) (161) (261) (361) (461) (561) (661) (761) and a heat source side heat exchanger (63) (163) (263) (363) (463) (563) (663) (763), and the heat source side refrigerant circuit being used as a heat source by both the first utilization side refrigerant circuit and the second utilization side refrigerant circuits, wherein

the first utilization side refrigerant circuit is being connected to a discharge gas connection pipe (8) (108) (208) (308) (408) (508) (608) (708) connected to a discharge side of the compression mechanism, and is being connected to an inlet gas connection pipe (9)

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(109) (209) (309) (409) (509) (609) (709) connected to an inlet side of the compression mechanism[[;]],

the air conditioning system can first utilization side refrigerant circuit being configured to supply a room with air that passed through the adsorbent heat exchanger, and the air conditioning system can second utilization side refrigerant circuits being configured to supply a room with air that passed through the air heat exchanger.

4. (Currently Amended) The air conditioning system (1) (101) (401) (501) according to claim 2 or claim 3, wherein

the second utilization side refrigerant circuits (10e, 10d) (110e, 110d) (410e, 410d) (510e, 510d) are connected to a liquid connection pipe (7) (107) (407) (507) that is connected to a liquid side of the heat source side heat exchanger (63) (163) (463) (563), and also switchably connected to the discharge gas connection pipe (8) (108) (408) (508) and the inlet gas connection pipe (9) (109) (409) (509) through a switching mechanism (71, 81) (171, 181) (471, 481) (571, 581).

5. (Currently Amended) The air conditioning system (201) (301) (601) (701) according to claim 2 or 3, wherein

the second utilization side refrigerant circuits (210e, 210d) (310e, 310d) (610e, 610d) (710e, 710d) are connected to a liquid connection pipe (207) (307) (607) (707) connected to a liquid side of the heat source side heat exchanger (263) (363) (663) (763), and are connected to the inlet gas connection pipe (209) (309) (609) (709).

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6. (Currently Amended) The air conditioning system (101) (301) (501) (701) according to any one of claims claim 2 to 5, wherein

the first utilization side refrigerant <u>circuits</u> eireuit (110a, 110b) (310a, 310b) (510a, 510b) (710a, 710b) and the second utilization side refrigerant <u>circuits</u> eireuit (110e, 110d) (310e, 310d) (510e, 510d) (710e, 710d) constitute an integrated utilization unit (102, 103) (302, 303) (502, 503) (702, 703).

7. (Currently Amended) The air conditioning system (101) (301) (501) (701) according to claim 6, wherein

the utilization unit (102, 103) (302, 303) (502, 503) (702, 703) can is configured to supply a room with air that was dehumidified or humidified in the adsorbent heat exchanger (122, 123, 132, 133) (322, 323, 332, 333) (522, 523, 532, 533) (722, 723, 732, 733).

8. (Currently Amended) The air conditioning system (101) (301) (501) (701) according to claim 6, wherein

the utilization unit (102, 103) (302, 303) (502, 503) (702, 703) can is configured to exchange heat through the air heat exchanger (142, 152) (342, 352) (542, 552) (742, 752) between refrigerant and air that was dehumidified of or humidified in the adsorbent heat exchanger (122, 123, 132, 133) (322, 323, 332, 333) (522, 523, 532, 533) (722, 723, 732, 733).

9. (Currently Amended) The air conditioning system (1)-(101) (201) (301) according to any one of claim 2 to claim 8, wherein

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the air conditioning system is configured to calculate a required latent heat treatment capacity value (Δh) and a required sensible heat treatment capacity value (ΔT) in order to control the <u>an</u> operational capacity of the compression mechanism (61) (161) (261) (361) based on the <u>a</u> required latent heat treatment capacity value and the <u>a</u> required sensible heat treatment capacity value.

10. (Currently Amended) The air conditioning system (1) (101) (201) (301) according to claim 9, wherein

the air conditioning system is configured to calculate a target evaporation temperature (TeS) and a target condensation temperature (TeS) of the system as a whole based on the required latent heat treatment capacity value (Δh) and the required sensible heat treatment capacity value (ΔT) in order to control the operational capacity of the compression mechanism (61) (161) (261) (361) based on the <u>a</u> target evaporation temperature and the <u>a</u> target condensation temperature.

11. (Currently Amended) The air conditioning system (1) (101) (201) (301) according to claim 10, wherein

the air conditioning system is configured to calculate an evaporation temperature difference (ATe) between the target evaporation temperature (TeS) and the an evaporation temperature (Te) and to calculate a condensation temperature difference (ATe) between the target condensation temperature (TeS) and the a condensation temperature (Te) in order to control the operational capacity of the compression mechanism (61) (161) (261) (361) based on the evaporation temperature difference and the condensation temperature difference.

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12. (Currently Amended) The air conditioning system (1) (101) (201) (301) according to any one of claims claim 9 to 11, wherein

a switching time interval between the adsorption process and the regeneration process in the adsorbent heat exchanger (22, 23, 32, 33) (122, 123, 132, 133) (322, 323, 332, 333) ean be changed is changeable.

13. (Currently Amended) The air conditioning system (1) (101) (201) (301) according to any one of claims claim 1 to 12, wherein

at system startup, a room is supplied with air that passed through the air heat exchanger (42, 52) (142, 152) (242, 252) (342, 352), and outdoor air is prevented from passing through the adsorbent heat exchanger (22, 23, 32, 33) (122, 123, 132, 133) (322, 323, 332, 333).

14. (Currently Amended) The air conditioning system (1) (101) (201) (301) according to any one of claims claim 1 to 12, wherein

at system startup, in a state in which switching between the adsorption process and the regeneration process in the plurality of adsorbent heat exchangers (22, 23, 32, 33) (122, 123, 132, 133) (322, 323, 332, 333) is stopped, outdoor air is passed through one of the plurality of adsorbent heat exchangers and then is exhausted to the outside, and also room air is passed through an adsorbent heat exchanger among the plurality of adsorbent heat exchangers, besides the one through which the outdoor air passed, and then is supplied to a room again.

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15. (Currently Amended) The air conditioning system (1) (101) (201) (301) according to any one of claims claim 1 to 12, wherein

at system startup, a switching time interval between the adsorption process and the regeneration process in the adsorbent heat exchanger (22, 23, 32, 33) (122, 123, 132, 133) (322, 323, 332, 333) is made longer than that during normal operation.

16. (Currently Amended) The air conditioning system (1) (101) (201) (301) according to any one of claims claim 13 to 15, wherein

the <u>a</u> system startup operation is terminated after a predetermined period of time elapsed since system startup.

17. (Currently Amended) The air conditioning system (1) (101) (201) (301) according to any one of claims claim 13 to 15, wherein

the <u>a</u> system startup operation is terminated after a temperature difference between the <u>a</u> target temperature of room air and the <u>a</u> temperature of room air is equal to or below a predetermined temperature difference.

18. (Currently Amended) The air conditioning system (1) (101) (201) (301) according to any one of claims claim 13 to 17, wherein

before the <u>a</u> system startup operation starts, whether or not a temperature difference between the <u>a</u> target temperature of room air and the <u>a</u> temperature of room air is equal to or below a predetermined temperature difference is determined, and

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when the temperature difference between the target temperature of room air and the temperature of room air is equal to or below a predetermined temperature, the system startup operation is prevented from being performed.

19. (Currently Amended) The air conditioning system (401) (501) (601) (701) according to any one of claims claim 2 to 8, further comprising

<u>a</u> pressure control mechanism (473, 483) (573, 583) (673, 683) (773, 783) connected to a gas side of the air heat exchanger (442, 452) (542, 552) (642, 652) (742, 752) and configured to control the <u>an</u> evaporation pressure of refrigerant in the air heat exchanger when the air heat exchanger is caused to function as an evaporator that evaporates refrigerant.

20. (Currently Amended) The air conditioning system (401) (501) (601) (701) according to claim 19, wherein

the evaporation pressure of refrigerant, when the air heat exchangers (442, 452) (542, 552) (642, 652) (742, 752) are exchanger is caused to function as an evaporator that evaporates refrigerant, the evaporation pressure of refrigerant is controlled by the pressure control mechanism (473, 483) (573, 583) (673, 683) (773, 783), based on the a dew point temperature of room air.

21. (Currently Amended) The air conditioning system (401) (501) (601) (701) according to claim 20, <u>further</u> comprising

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a pressure detection mechanism (474, 484) (574, 584) (674, 684) (774, 784) configured to detect the <u>a</u> refrigerant pressure in the air heat exchanger (442, 452) (542, 552) (642, 652) (742, 752) and an evaporation pressure of refrigerant, wherein

the air conditioning system calculates a target evaporation pressure value (P3) based on the dew point temperature of room air and uses the pressure control mechanism (473, 483) (573, 583) (673, 683) (773, 783) to control the evaporation pressure of refrigerant, which was detected by the pressure detection mechanism, to be equal to or higher than the target evaporation pressure.

22. (Currently Amended) The air conditioning system (401) (501) (601) (701) according to claim 21, <u>further</u> comprising

<u>a plurality of</u> condensation detection mechanisms (446, 456) (546, 556) (646, 656) (746, 756) configured to detect the <u>a</u> presence of condensation in the air heat exchangers (442, 452) (542, 552) (642, 652) (742, 752), wherein

when condensation is detected by the condensation detection mechanism, the target evaporation pressure value (P3) is changed.

23. (Currently Amended) The air conditioning system (401) (501) (601) (701) according to claim 21, <u>further</u> comprising

a condensation detection mechanism (446, 456) (546, 556) (646, 656) (746, 756) configured to detect the <u>a</u> presence of condensation in the air heat exchanger (442, 452) (542, 552) (642, 652) (742, 752), wherein

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when condensation is detected by the condensation detection mechanisms mechanism, the compression mechanism (461) (561) (661) (761) is stopped.

24. (Currently Amended) The air conditioning system (401) (501) (601) (701) according to claim 21, <u>further</u> comprising

a condensation detection mechanism (446, 456) (546, 556) (646, 656) (746, 756) configured to detect the a presence of condensation in the air heat exchanger (442, 452) (542, 552) (642, 652) (742, 752), wherein,

the second utilization side refrigerant circuit (410e, 410d) (510e, 510d) (610e, 610d) (710e, 710d) comprises includes an utilization side expansion valve (441, 451) (541, 551) (641, 651) (741, 751) connected to a liquid side of the air heat exchangers, and when condensation is detected by the condensation detection mechanism, the utilization side expansion valve is closed.

25. (Currently Amended) The air conditioning system (401) (501) (601) (701) according to any one of claims claim 2 to 8 and 19 to 24, wherein

a switching time interval between the adsorption process and the regeneration process in the adsorbent heat exchanger (22, 23, 32, 33) (522, 523, 532, 533) (722, 723, 732, 733) can be changed is changeable.

26. (Currently Amended) The air conditioning system (401) (501) (601) (701) according to any one of claims claim 19 to 25, wherein

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at system startup, treatment of the <u>a</u> latent heat load in a room by the first utilization side refrigerant circuit (410a, 410b) (510a, 510b) (610a, 610b) (710a, 710b) is given priority over treatment of the <u>a</u> sensible heat load in a room by the second utilization side refrigerant circuit (410e, 410d) (510e, 510d) (610e, 610d) (710e, 710d).

27. (Currently Amended) The air conditioning system (401) (501) (601) (701) according to claim 26, wherein

at system startup, treatment of the sensible heat load in a room by the second utilization side refrigerant circuit (410e, 410d) (510e, 510d) (610e, 610d) (710e, 710d) is stopped until the <u>a</u> dew point temperature of room air is equal to or below the <u>a</u> target dew point temperature.

28. (Currently Amended) The air conditioning system (401) (501) (601) (701) according to claim 26, wherein

at system startup, treatment of the sensible heat load in a room by the second utilization side refrigerant circuit (410c, 410d) (510c, 510d) (610c, 610d) (710c, 710d) is stopped until the an absolute humidity of room air is equal to or below the a target absolute humidity.

29. (Currently Amended) The air conditioning system (401) (501) (601) (701) according to any one of claims claim 26 to 28, wherein

at system startup, outdoor air is passed through an one of the adsorbent heat exchanger, which exchangers that is performing the a regeneration process, among the

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plurality of adsorbent heat exchangers (22, 23, 32, 33) (522, 523, 532, 533) (722, 723, 732, 733), and then is exhausted to the outside, and also, then room air is passed through an one of the adsorbent heat exchanger, whichever exchangers that is performing the adsorption process, among the plurality of adsorbent heat exchangers, and then is again supplied to a room.

30. (Currently Amended) The air conditioning system (401) (501) (601) (701) according to any one of claims claim 26 to 29, wherein

before starting the <u>a</u> system startup operation, whether or not a dew point temperature difference between the <u>a</u> target dew point temperature of room air and the <u>a</u> dew point temperature of the room air is equal to or below a predetermined dew point temperature difference is determined, and

when a the dew point temperature difference between the target dew point temperature of room air and the dew point temperature of room air is equal to or below a predetermined dew point temperature difference, the startup operation is prevented from being performed.

31. (Currently Amended) The air conditioning system (401) (501) (601) (701) according to any one of claims claim 26 to 29, wherein

before starting the <u>a</u> system startup operation, whether or not an absolute humidity difference between the <u>a</u> target absolute humidity of room air and the <u>an</u> absolute humidity of the room air is equal to or below a predetermined absolute humidity difference determined, and

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when an the absolute humidity difference between the target absolute humidity of room air and the absolute humidity of room air is equal to or below a predetermined absolute humidity difference, the system startup operation is prevented from being performed.

32. (New) The air conditioning system according to claim 3, wherein the second utilization side refrigerant circuits are connected to a liquid connection pipe that is connected to a liquid side of the heat source side heat exchanger, and switchably connected to the discharge gas connection pipe and the inlet gas connection pipe through a switching mechanism.

- 33. (New) The air conditioning system according to claim 3, wherein the second utilization side refrigerant circuits are connected to a liquid connection pipe connected to a liquid side of the heat source side heat exchanger, and are connected to the inlet gas connection pipe.
- 34. (New) The air conditioning system according to claim 3, wherein the air conditioning system is configured to calculate a required latent heat treatment capacity value and a required sensible heat treatment capacity value in order to control an operational capacity of the compression mechanism based on a required latent heat treatment capacity value and a required sensible heat treatment capacity value.
 - 35. (New) The air conditioning system according to claim 3, wherein

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at system startup, a room is supplied with air that passed through the air heat exchanger, and outdoor air is prevented from passing through the adsorbent heat exchanger.

36. (New) The air conditioning system according to claim 3, wherein at system startup, in a state in which switching between the adsorption process and the regeneration process in the plurality of adsorbent heat exchangers is stopped, outdoor air is passed through one of the plurality of adsorbent heat exchangers and then is exhausted to the outside, and also room air is passed through an adsorbent heat exchanger among the plurality of adsorbent heat exchangers, besides the one through which the outdoor air passed, and then is supplied to a room again.

- 37. (New) The air conditioning system according to claim 3, further comprising a pressure control mechanism connected to a gas side of the air heat exchanger and configured to control an evaporation pressure of refrigerant in the air heat exchanger when the air heat exchanger is caused to function as an evaporator that evaporates refrigerant.
- 38. (New) The air conditioning system according to claim 3, wherein a switching time interval between the adsorption process and the regeneration process in the adsorbent heat exchanger is changeable.